# **Coupled Oscillators**

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## I. Connected Masses $(m(2) ext{ and } k(3))$



 $k_{3}$  Two masses can move horizontally. They are connected to each other and the walls by 3 springs as shown.

consider the forces on mass 1:

$$egin{array}{l} F_{ ext{on mass }1} = -k_1 x_1 + k_2 \left( x_2 - x_1 
ight) \ = - \left( k_1 + k_2 
ight) x_1 + k_2 x_2 \end{array}$$

consider the forces on mass 2:

$$egin{array}{l} F_{ ext{on mass }2} &= -k_3 x_2 - k_2 \left( x_2 - x_1 
ight) \ &= k_2 x_1 - \left( k_2 + k_3 
ight) x_2 \end{array}$$

Two masses,  $m_1$  and  $m_2$  connected via three springs.

This leads to 2 eqs. of motion.

$$egin{aligned} m_1 \ddot{x}_1 &= -\left(k_1 + k_2
ight) x_1 + k_2 x_2 \ m_2 \ddot{x}_2 &= k_2 x_1 - \left(k_2 + k_3
ight) x_2 \end{aligned}$$

#### I.I Matrix Form

With the following definitions:

$$\mathbf{x} = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \tag{1}$$

$$\mathbf{M} = \begin{bmatrix} m_1 & 0\\ 0 & m_2 \end{bmatrix} \tag{2}$$

$$\mathbf{K} = \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix}$$
(3)

We can write:

$$\mathbf{M}\ddot{\mathbf{x}} = -\mathbf{K}\mathbf{x} \tag{4}$$

How to solve this?

Assume a complex solution:

$$z_{1} = x_{1}(t) + iy_{1}(t) = \alpha_{1}e^{i(\omega t - \delta_{1})}$$

$$= \alpha_{1}e^{-i\delta_{1}}e^{i\omega t} = a_{1}e^{i\omega t}$$

$$z_{2} = \dots = a_{2}e^{i\omega t}$$

$$\mathbf{z}(t) = \begin{bmatrix} z_{1}(t) \\ z_{2}(t) \end{bmatrix} = \begin{bmatrix} a_{1} \\ a_{2} \end{bmatrix} e^{i\omega t} = \mathbf{a}e^{i\omega t}$$
(5)

note:  $\mathbf{x}(t) = \operatorname{Re} \mathbf{z}(t)$ 

Start with the equation of motion:

$$\mathbf{M}\ddot{\mathbf{x}} = -\mathbf{K}\mathbf{x} \tag{6}$$

Replace  $\mathbf{x}$  with our complex  $\mathbf{z}$  and we obtain:

$$-\omega^2 \mathbf{M} \mathbf{a} e^{i\omega t} = -\mathbf{K} \mathbf{a} e^{i\omega t}$$
(7)

Canceling the exponential terms and re-arranging:

$$\left(\mathbf{K} - \omega^2 \mathbf{M}\right) \mathbf{a} = 0 \tag{8}$$

Now we ask, what do solutions to this equation look like?

Clearly, a could be zero, then we have the so-called trivial solution, i.e. nothing moves. Boring.

But if :

$$\det\left(\mathbf{K} - \omega^2 \mathbf{M}\right) = 0 \tag{9}$$

then we might have more interesting solutions.

## 2. Equal m and k

Let's try a simple case, where the two masses are equal and the spring constants are all the same:

$$\mathbf{M} = \begin{bmatrix} m & 0 \\ 0 & m \end{bmatrix} \text{ and } \mathbf{K} = \begin{bmatrix} 2k & -k \\ -k & 2k \end{bmatrix}$$
(10)

Our Matrix is thus:

$$\left(\mathbf{K} - \omega^{2}\mathbf{M}\right) = \begin{bmatrix} 2k - m\omega^{2} & -k \\ -k & 2k - m\omega^{2} \end{bmatrix}$$
(11)

Taking the determinant of this yields:

$$\det \left(\mathbf{K} - \omega^2 \mathbf{M}\right) = \left(2k - m\omega^2\right)^2 - k^2 \tag{12}$$

$$= \left(k - m\omega^2\right) \left(3k - m\omega^2\right) \tag{13}$$

There will be two frequencies that create a 0

$$\left(k-m\omega^2
ight)\left(3k-m\omega^2
ight)$$
 (14)

$$\omega = \sqrt{\frac{k}{m}} = \omega_1 \text{ and } \omega = \sqrt{\frac{3k}{m}} = \omega_2$$
 (15)

Thus we have two non-trivial solutions. Our next step is to find the motions that occur at these frequencies

2.2 Normal modes

## 3. First Normal Mode

$$\left(\mathbf{K} - \omega^2 \mathbf{M}\right) \mathbf{a} = 0 \tag{16}$$

Start with  $\omega_1=\sqrt{k/m}$ 

$$\left(\mathbf{K} - \omega_1^2 \mathbf{M}\right) = \begin{bmatrix} k & -k \\ -k & k \end{bmatrix}$$
(17)

$$\begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} a_1 \\ a_2 \end{bmatrix} = 0 \tag{18}$$

which leads to two equations:

$$a_1 - a_2 = 0$$
  
 $-a_1 + a_2 = 0$ 

that both imply  $a_1 = a_2$ 

if we say:

 $a_1=a_2=Ae^{-i\delta}$ 

then we can write our complex solution 
$$\mathbf{z}(t)$$
 as:

$$\mathbf{z}(t) = \begin{bmatrix} a_1 \\ a_2 \end{bmatrix} e^{i\omega_1 t} = \begin{bmatrix} A \\ A \end{bmatrix} e^{i(\omega_1 t - \delta)}$$
(19)

The actual motion of the masses is then described by the *real* part of the above:

$$\mathbf{x}(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} A \\ A \end{bmatrix} \cos(\omega_1 t - \delta)$$

$$x_1(t) = A \cos(\omega_1 t - \delta)$$

$$x_2(t) = A \cos(\omega_1 t - \delta)$$
(20)

## 4. Second Normal Mode

Now Choose  $\omega_2=\sqrt{3k/m}$ 

$$\left(\mathbf{K} - \omega_2^2 \mathbf{M}\right) = \begin{bmatrix} -k & -k \\ -k & -k \end{bmatrix}$$
 (21)

$$\begin{bmatrix} 1 & 1 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} a_1 \\ a_2 \end{bmatrix} = 0$$
(22)

which leads to two equations:

$$a_1 + a_2 = 0$$

which implies  $a_1=-a_2$ 

if we say:

 $a_1=-a_2=Ae^{-i\delta}$ 

then we can write our complex solution  $\mathbf{z}(t)$  as:

$$\mathbf{z}(t) = \begin{bmatrix} a_1 \\ a_2 \end{bmatrix} e^{i\omega_2 t} = \begin{bmatrix} A \\ -A \end{bmatrix} e^{i(\omega_2 t - \delta)}$$
(23)

The motion of the masses is then described by the *real* part of the above:

$$\mathbf{x}(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} A \\ -A \end{bmatrix} \cos(\omega_2 t - \delta)$$

$$x_1(t) = A \cos(\omega_2 t - \delta)$$

$$x_2(t) = -A \cos(\omega_2 t - \delta)$$
(24)





#### 4.3 General Solution

$$egin{aligned} \mathbf{x}(t) &= A_1 egin{bmatrix} 1 \ 1 \end{bmatrix} \cos{(\omega_1 t - \delta_1)} \ \mathbf{x}(t) &= A_2 egin{bmatrix} 1 \ -1 \end{bmatrix} \cos{(\omega_2 t - \delta_2)} \end{aligned}$$

These are both solutions to

$$\mathbf{M}\ddot{\mathbf{x}} = -\mathbf{K}\mathbf{x} \tag{25}$$

which means their sum is also also a solution.

$$\mathbf{x}(t) = A_1 \begin{bmatrix} 1\\1 \end{bmatrix} \cos\left(\omega_1 t - \delta_1\right) + A_2 \begin{bmatrix} 1\\-1 \end{bmatrix} \cos\left(\omega_2 t - \delta_2\right)$$
(26)



An arbitrary plot for the general solution.



#### 4.4 Physical Example



A Carbon Dioxide Molecule

## 5. Larger systems

Now, let's try it with the Lagrangian formulation:

$$T = \frac{1}{2}m(\dot{x}_1^2 + \dot{x}_2^2 + \dot{x}_3^2) \tag{27}$$

More masses?

$$U = rac{1}{2}kx_1^2 + rac{1}{2}k(x_2 - x_1)^2 + rac{1}{2}k(x_3 - x_2)^2 + rac{1}{2}kx_3^2$$
 (28)

Thus our  $oldsymbol{L}$  is:

$$L=rac{1}{2}m(\dot{x}_1^2+\dot{x}_2^2+\dot{x}_2^2)-rac{1}{2}kx_1^2-rac{1}{2}k(x_2-x_1)^2-rac{1}{2}k(x_3-x_2)^2-rac{1}{2}kx_3^2$$
 (29)

After solving the Euler-Lagrange for this system:

$$egin{aligned} & m\ddot{x}_1 = -kx_1 + k(x_2 - x_1) \ & m\ddot{x}_2 = -k(x_2 - x_1) + k(x_3 - x_2) \ & m\ddot{x}_3 = -kx_3 - k(x_3 - x_2) \end{aligned}$$

Adopt exponential solutions:

$$egin{aligned} x_1 &= b_1 e^{i\omega t} \ x_2 &= b_2 e^{i\omega t} \ x_3 &= b_3 e^{i\omega t} \end{aligned}$$

Which leads to a 3 x 3 matrix version:

$$\begin{pmatrix} -m\omega^{2} + 2k & -k & 0\\ -k & -m\omega^{2} + 2k & -k\\ 0 & -k & -m\omega^{2} + 2k \end{pmatrix} \begin{pmatrix} b_{1}\\ b_{2}\\ b_{3} \end{pmatrix} = 0$$
(30)

Taking the determinant of this and setting equal to zero:

$$\begin{vmatrix} -m\omega^{2} + 2k & -k & 0\\ -k & -m\omega^{2} + 2k & -k\\ 0 & -k & -m\omega^{2} + 2k \end{vmatrix} = 0$$
(31)

Will lead to a polynomial

$$(-m\omega^2 + 2k)[(-m\omega^2 + 2k)^2 - k^2] + k(-k(-m\omega^2 + 2k)) = 0$$
(32)

This can be factored:

$$(-m\omega^2 + 2k)[(-m\omega^2 + 2k)^2 - 2k^2] = 0$$
(33)

if

$$(-m\omega^2 + 2k) = 0 \tag{34}$$

then we have a root:

$$\omega_1 = \sqrt{\frac{2k}{m}} \tag{35}$$

likewise: if

$$[(-m\omega^2 + 2k)^2 - 2k^2] = 0$$
(36)

then:

$$\omega_{2} = \sqrt{\frac{(2-\sqrt{2})k}{m}} \text{ and } \omega_{3} = \sqrt{\frac{(2+\sqrt{2})k}{m}}$$

$$b_{2} = 0, \quad b_{3} = -b_{1} \quad \text{for } \omega = \omega_{1}$$

$$b_{3} = b_{1}, \quad b_{2} = \sqrt{2}b_{1} \quad \text{for } \omega = \omega_{2}$$

$$b_{3} = b_{1}, \quad b_{2} = -\sqrt{2}b_{1} \quad \text{for } \omega = \omega_{3}$$
(37)

## 6. The continuum limit

$$L = T - U = \sum_{i} rac{1}{2} m \dot{x}_{i}^{2} - rac{1}{2} \sum_{i} k (x_{i+1} - x_{i})^{2}$$
 (38)

The lagrangian for a mass  $\boldsymbol{m}$  is then:

$$m\ddot{x}_i - k(x_{i+1} - x_i) + k(x_i - x_{i-1}) = 0$$
(39)

or:

$$k(x_{i+1} - x_i) - k(x_i - x_{i-1}) = m\ddot{x}_i$$
(40)

Define a mass per unit length:

$$\mu = \frac{m}{a} \tag{41}$$

as well as the Young's Modulus:

$$Y = \frac{\text{stress}}{\text{strain}} \tag{42}$$

$$stress = T = k(x_{i+1} - x) \tag{43}$$

and

$$\text{strain} = \frac{x_{i+1} - x_i}{a} \tag{44}$$

Stress: is tension / Strain is extension per unit length

Thus:

$$Y = \frac{k(x_{i+1} - x_i)}{(x_{i+1} - x_i)/a} = ka$$
(45)

Now, let a 
ightarrow 0

$$\mu \ddot{x}_i - \frac{Y}{a} [(x_{i+1} - x_i) - (x_i - x_{i-1})] = 0$$
(46)

Call  $\eta(x,t)$  the displacement from equilibrium

$$\ddot{x}_i \to \left. \frac{\partial^2 \eta}{\partial t^2} \right|_{x_i}$$

$$\tag{47}$$

$$\frac{(x_{i+1} - x_i)}{a} \to \frac{\partial \eta}{\partial x}\Big|_{x_i + a}$$
(48)

$$\frac{(x_i - x_{i-1})}{a} \to \frac{\partial \eta}{\partial x}\Big|_{x_i}$$
(49)

From Taylor Series:

$$\left. \frac{\partial \eta}{\partial x} \right|_{x_i+a} = \left. \frac{\partial \eta}{\partial x} \right|_{x_i} + a \frac{\partial^2 \eta}{\partial x^2} \right|_{x_i+a}$$
(50)

### 6.5 Wave Equation:

$$\frac{\partial^2 \eta}{\partial x^2} - \frac{\mu}{Y} \frac{\partial^2 \eta}{\partial t^2} = 0$$
(51)

Replace the  $\mu/Y$  with the velocity,  $v^2$ .

$$\frac{\partial^2 \eta}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 \eta}{\partial t^2} = 0$$
 (52)